Final Examination
Thursday, August 9, 2018
9:30 am to $12: 00 \mathrm{pm}$
3109 Etcheverry Hall

## Closed Books and Closed Notes <br> For Full Credit Answer All Five Questions <br> Question 1 is worth 10 Points <br> Questions 2 and 4 are worth 20 Points Each Questions 3 and 5 are worth 30 Points Each

## Useful Formulae

For the corotational basis $\left\{\mathbf{e}_{x}, \mathbf{e}_{y}, \mathbf{e}_{z}=\mathbf{E}_{z}\right\}$ shown in the figures

$$
\begin{align*}
\mathbf{e}_{x} & =\cos (\theta) \mathbf{E}_{x}+\sin (\theta) \mathbf{E}_{y}, \\
\mathbf{e}_{y} & =\cos (\theta) \mathbf{E}_{y}-\sin (\theta) \mathbf{E}_{x} . \tag{1}
\end{align*}
$$

The following identity for the angular momentum of a rigid body relative to a point $P$ will also be useful:

$$
\begin{equation*}
\mathbf{H}_{P}=\mathbf{H}+\left(\overline{\mathbf{x}}-\mathbf{x}_{P}\right) \times m \overline{\mathbf{v}} . \tag{2}
\end{equation*}
$$

The expression for the inverse of the following matrix will be useful in answering one of the questions:

$$
\left[\begin{array}{cc}
m_{1} & m_{1}  \tag{3}\\
m_{1} & m_{1}+m_{2}
\end{array}\right]^{-1}=\left[\begin{array}{cc}
\frac{m_{1}+m_{2}}{m_{1} m_{2}} & -\frac{1}{m_{2}} \\
-\frac{1}{m_{2}} & \frac{1}{m_{2}}
\end{array}\right] .
$$

In computing components of moments, the following identity can be useful:

$$
\begin{equation*}
(\mathbf{a} \times \mathbf{b}) \cdot \mathbf{E}_{z}=\left(\mathbf{E}_{z} \times \mathbf{a}\right) \cdot \mathbf{b} \tag{4}
\end{equation*}
$$

Finally, recall that the work-energy theorem of a rigid body which is subject to a system of $K$ forces and a pure moment $\mathbf{M}_{p}$ is

$$
\begin{equation*}
\dot{T}=\sum_{i=1}^{K} \mathbf{F}_{i} \cdot \mathbf{v}_{i}+\mathbf{M}_{p} \cdot \boldsymbol{\omega} . \tag{5}
\end{equation*}
$$

Here, $\mathbf{v}_{i}$ is the velocity vector of the point where the force $\mathbf{F}_{i}$ is applied.

## Question 1

A Rigid Body Tumbling through the Air (10 Points)

As shown in Figure 1, a slender rigid rod of length $\ell$, mass $m$, and moment of inertia $I_{z z}=\frac{m \ell^{2}}{12}$ is thrown into the air. A vertical gravitational force $-m g \mathbf{E}_{y}$ acts on the rod and drag is ignored. The position vector of the center of mass $C$ of the rod is described using a set of Cartesian coordinates,

$$
\begin{equation*}
\overline{\mathbf{x}}=x \mathbf{E}_{x}+y \mathbf{E}_{y}+z \mathbf{E}_{z}, \tag{6}
\end{equation*}
$$

and the angular velocity vector of the rod has the representation:

$$
\begin{equation*}
\boldsymbol{\omega}=\dot{\theta} \mathbf{E}_{z} . \tag{7}
\end{equation*}
$$



Figure 1: A slender rod moving through the air.
Suppose the rod is thrown into the air with the following initial conditions:
$\overline{\mathbf{x}}(t=0)=x_{0} \mathbf{E}_{x}+y_{0} \mathbf{E}_{y}, \quad \overline{\mathbf{v}}(t=0)=v_{0_{x}} \mathbf{E}_{x}+v_{0_{y}} \mathbf{E}_{y}, \quad \theta(t=0)=\theta_{0}, \quad \dot{\theta}(t=0)=\omega_{0}$.
Determine the resulting motion $\overline{\mathbf{x}}(t)$ and $\theta(t)$ of the rod.

## Question 2 <br> Tipping Points (20 Points)

As shown in Figure 2, a crate is being moved up and down an incline with the help of a force $P_{0} \mathbf{E}_{x}$. The crate is modeled as a rigid body of mass $m$ and moment of inertia $I_{z z}$ relative to its center of mass $C$ which has two contact points $A$ and $B$ with the incline. During its motion, a vertical gravitational force acts on the crate and the center of mass $C$ of the crate has a position vector

$$
\begin{equation*}
\overline{\mathbf{x}}=x \mathbf{E}_{x}+y_{0} \mathbf{E}_{y}, \tag{9}
\end{equation*}
$$

where $y_{0}$ is a constant.


Figure 2: A crate being pushed up an incline by a constant force $P_{0} \mathbf{E}_{x}$.
(a) (5 Points) Establish expressions for the angular momentum $\mathbf{H}_{O}$ and kinetic energy $T$ for the rigid body assuming that it is not rotating.
(b) (5 Points) Assuming that both $A$ and $B$ are in contact with the incline, draw a freebody diagram of the rigid body.
(c) (5 Points) Using balances of linear and angular momentum, assuming that both $A$ and $B$ are in contact with the incline show that the normal forces acting at $A$ and $B$ are, respectively,

$$
\begin{equation*}
\mathbf{N}_{A}=\left(\frac{m g}{2} \cos (\beta)-?\right) \mathbf{E}_{y}, \quad \mathbf{N}_{B}=\left(\frac{m g}{2} \cos (\beta)+?\right) \mathbf{E}_{y} \tag{10}
\end{equation*}
$$

and

$$
\begin{equation*}
m \ddot{x}=P_{0}-m g \sin (\beta) . \tag{11}
\end{equation*}
$$

For full credit supply the missing terms.
(d) $(5$ Points $)$ To prevent the crate from tipping over, show that $P_{0}$ must be limited as follows:

$$
\begin{equation*}
-\frac{b}{\ell} \cos (\beta) \leq \frac{P_{0}}{m g} \leq \frac{b}{\ell} \cos (\beta) . \tag{12}
\end{equation*}
$$

## Question 3

A Suspended Rigid Body
(30 Points)
As shown in Figure 3, a thin uniform rigid plate of mass $m_{1}$, width $2 w$ and breadth $2 b$ is pinjointed at $O$. A particle of mass $m_{2}$ is rigidly attached to the plate at the location shown in the figure. During the motion of the composite rigid body of mass $m_{1}+m_{2}$, a vertical gravitational force also acts on the rigid body along with a force $\mathbf{P}$ which has a constant magnitude $P_{0}$. The direction of the force $\mathbf{P}$ is determined by the cable that is attached to a point $S$ on the plate. The position vectors of the center of mass $C_{1}$ of the plate and the points $S$ and $A$ relative to the point $O$, and the angular momentum of the plate relative to $C_{1}$ have the representations

$$
\begin{equation*}
\overline{\mathbf{x}}_{1}=w \mathbf{e}_{x}-b \mathbf{e}_{y}, \quad \mathbf{x}_{S}=2 w \mathbf{e}_{x}, \quad \mathbf{x}_{A}=2 w \mathbf{E}_{y}, \quad \mathbf{H}_{1}=\left(I_{z z}=\frac{m_{1}}{3}\left(w^{2}+b^{2}\right)\right) \dot{\theta} \mathbf{E}_{z} . \tag{13}
\end{equation*}
$$



Figure 3: A rigid body of mass $m_{1}+m_{2}$ is free to rotate about $O$ and is subject to a gravitational force.
(a) (5 Points) Establish expressions for the angular momentum $\mathbf{H}_{O}$ and kinetic energy $T$ of the rigid body of mass $m_{1}+m_{2}$ when it is rotating about $O$.
(b) $(5$ Points) Draw a free-body diagram of the rigid body when it is rotating about $O$.
(c) (5 Points) Show that the following differential equation governs $\theta$ when the body is rotating about $O$ :

$$
\begin{equation*}
\left(\frac{4 m_{1}}{3}\left(w^{2}+b^{2}\right)+4 m_{2}\left(w^{2}+b^{2}\right)\right) \ddot{\theta}=? ? ? ? g+? ? ? ? ? P_{0} . \tag{14}
\end{equation*}
$$

For full credit, supply the missing terms (some of which may be negative).
(d) (5 Points) Prove that the force applied by the cable to the rigid body is conservative with a potential energy

$$
\begin{equation*}
U_{\mathbf{P}}=P_{0}\left\|\mathbf{x}_{A}-\mathbf{x}_{S}\right\| . \tag{15}
\end{equation*}
$$

(e) (5 Points) Starting from the work-energy theorem (5), prove that the total energy $E$ of the rigid body is conserved. For full credit, supply an expression for the total energy $E$.
(f) (5 Points) Suppose that the rigid body is released from rest with $\theta=0$. Determine the minimum value of the force $P_{0}$ needed to raise the body to $\theta=\frac{\pi}{2}$.

## Question 4

A Particle in a Slotted Disk (20 Points)
As shown in Figure 4, a particle of mass $m_{1}$ is attached by a spring to a circular disk of mass $m_{2}$. The particle is free to move in a smooth grove milled on the surface of the circular disk. The disk is fixed at its center point $O$ and is free to rotate about a vertical $\mathbf{E}_{z}$ axis.

The position vector of the particle relative to the center $O$ of the disk is

$$
\begin{equation*}
\mathbf{x}_{1}=x \mathbf{e}_{x} \tag{16}
\end{equation*}
$$

The moment of inertia of the disk is $I_{O_{z z}}$ and the stiffness and unstretched length of the spring are $K$ and $\ell_{0}$, respectively.


Figure 4: A particle of mass $m_{1}$ is free to move in a smooth slot on a circular disk of mass $m_{2}$.
(a) (5 Points) Show that the kinetic energy and angular momentum relative to $O$ of the particle-disk system is

$$
\begin{align*}
\mathbf{H}_{O} & =\left(I_{O_{z z}}+m_{1} x^{2}\right) \dot{\theta} \mathbf{E}_{z}, \\
T & =\frac{m_{1}}{2} \dot{x}^{2}+\frac{1}{2}\left(I_{O_{z z}}+m_{1} x^{2}\right) \dot{\theta}^{2} . \tag{17}
\end{align*}
$$

(b) (5 Points) Draw a free-body diagram of the particle and a free-body diagram of the circular disk. For full credit, provide a clear expression for the spring force.
(c) (5 Points) Show that the equations governing the motion of the disk-particle system are

$$
\begin{equation*}
I_{O_{z z}} \ddot{\theta}+? ?+? ? ?=0, \quad m_{1}\left(\ddot{x}-x \dot{\theta}^{2}\right)=? ? ? ? . \tag{18}
\end{equation*}
$$

For full credit supply the missing terms in these equations.
(d) (5 Points) Give an expression for the total energy $E$ of the system and verify either with the help of (18) or using a work-energy theorem, that $\dot{E}=0$.

## Question 5 <br> Sliding of a Rigid Body (30 Points)

As shown in Figure 5, a circular cylinder of radius $R$, mass $m_{1}$, and moment of inertia (relative to its center of mass $\left.C_{1}\right) I_{z z}$ is free to move atop a cart of mass $m_{2}$. A constant force $F_{0} \mathbf{E}_{x}$ acts on the cart. The position vectors of the centers of mass $C_{1}$ and $C_{2}$ have the representations

$$
\begin{equation*}
\overline{\mathbf{x}}_{1}=\left(x_{1}+x_{2}\right) \mathbf{E}_{x}+y_{0} \mathbf{E}_{y}, \quad \overline{\mathbf{x}}_{2}=x_{2} \mathbf{E}_{x} \tag{19}
\end{equation*}
$$

where $y_{0}$ is a constant. The point $P$ is the instantaneous point of contact of the cylinder and the cart.


Figure 5: A rigid cylinder of mass $m_{1}$ and radius $R$ moving on a cart. A constant force $F_{0} \mathbf{E}_{x}$ acts on the cart.
(a) (5 Points) With the help of the identity $\mathbf{v}_{2}=\mathbf{v}_{1}+\boldsymbol{\omega} \times\left(\mathbf{x}_{2}-\mathbf{x}_{1}\right)$ applied to two points on the cylinder, show that slip speed $v_{\text {rel }}$ of the cylinder relative to the cart is

$$
\begin{equation*}
v_{\mathrm{rel}}=\dot{x}_{1}+R \dot{\theta} \tag{20}
\end{equation*}
$$

where $\boldsymbol{\omega}=\dot{\theta} \mathbf{E}_{z}$ is the angular velocity of the cylinder.
(b) (5 Points) Draw free-body diagrams of the cylinder and the cart. Assume that the cylinder is sliding on the cart. Provide a clear expression for the friction force $\mathbf{F}_{f}$ acting on the cylinder.
(c) (10 Points) Assume that the cylinder is sliding on the cart. Show that the equations of motion of the system can be expressed in the following fashion:

$$
\begin{align*}
{\left[\begin{array}{cc}
m_{1} & m_{1} \\
m_{1} & m_{1}+m_{2}
\end{array}\right]\left[\begin{array}{l}
\ddot{x}_{1} \\
\ddot{x}_{2}
\end{array}\right] } & =\left[\begin{array}{c}
-\mu_{k} m_{1} g ? ? \\
? ? ?
\end{array}\right], \\
I_{z z} \ddot{\theta} & =-\mu_{k} m_{1} g R ? ? . \tag{21}
\end{align*}
$$

For full credit supply the missing terms.
(d) (5 Points) Show that the slip speed of the cylinder relative to the cart is governed by the equation

$$
\begin{equation*}
\dot{v}_{\mathrm{rel}}=-\frac{F_{0}}{m_{2}}-\mu_{k} m_{1} g\left(\frac{R^{2}}{I_{z z}}+\frac{m_{1}+m_{2}}{m_{1} m_{2}}\right) \frac{v_{\mathrm{rel}}}{\left|v_{\mathrm{rel}}\right|} \tag{22}
\end{equation*}
$$

(e) (5 Points) With the help of a work-energy theorem, show that the total energy $E$ of the system decreases with time. In your solution, provide a clear expression for $E$

