# Final Examination <br> Thursday, August 15, 2013 <br> 10:00 am to $12: 30 \mathrm{pm}$ <br> 101 Barker Hall 

## Closed Books and Closed Notes <br> For Full Credit Answer All Four Questions

## Useful Formulae

For all the corotational bases shown in the figures

$$
\begin{align*}
& \mathbf{e}_{x}=\cos (\theta) \mathbf{E}_{x}+\sin (\theta) \mathbf{E}_{y}, \\
& \mathbf{e}_{y}=\cos (\theta) \mathbf{E}_{y}-\sin (\theta) \mathbf{E}_{x} . \tag{1}
\end{align*}
$$

The following identity for the angular momentum of a rigid body relative to a point $P$ will also be useful:

$$
\begin{equation*}
\mathbf{H}_{P}=\mathbf{H}+\left(\overline{\mathbf{x}}-\mathbf{x}_{P}\right) \times m \overline{\mathbf{v}} . \tag{2}
\end{equation*}
$$

In computing components of moments, the following identity can be useful:

$$
\begin{equation*}
(\mathbf{a} \times \mathbf{b}) \cdot \mathbf{E}_{z}=\left(\mathbf{E}_{z} \times \mathbf{a}\right) \cdot \mathbf{b} \tag{3}
\end{equation*}
$$

The following derivative is also recorded:

$$
\begin{equation*}
\frac{d}{d y}\left(\frac{a y}{b+c y^{2}}\right)=\frac{a\left(b-c y^{2}\right)}{\left(b+c y^{2}\right)^{2}} \tag{4}
\end{equation*}
$$

where $a, b$, and $c$ are constant.
Finally, recall that the work-energy theorem of a rigid body which is subject to a system of $K$ forces and a pure moment $\mathbf{M}_{p}$ is

$$
\begin{equation*}
\dot{T}=\sum_{i=1}^{K} \mathbf{F}_{i} \cdot \mathbf{v}_{i}+\mathbf{M}_{p} \cdot \boldsymbol{\omega} \tag{5}
\end{equation*}
$$

Here, $\mathbf{v}_{i}$ is the velocity vector of the point where the force $\mathbf{F}_{i}$ is applied.

## Question 1 <br> A Moving Van (25 Points)

As shown in Figure 1, a rigid body of mass $m$ is attached by a pin-joint at $A$ to the flat bed of a truck. The rotation of the body about $A$ is resisted by a torsional spring of stiffness $K$. The spring exerts a restoring moment $\mathbf{M}_{s}$ on the rigid body:

$$
\begin{equation*}
\mathbf{M}_{s}=-K \theta \mathbf{E}_{z} \tag{6}
\end{equation*}
$$

A vertical gravitational force also acts on the rigid body. The position vectors of the center of mass $C$ of the rigid body and the point $A$, and the angular momentum of the rigid body relative to its center of mass have the representations

$$
\begin{equation*}
\overline{\mathbf{x}}=\mathbf{x}_{A}+\ell \mathbf{e}_{y}, \quad \mathbf{x}_{A}=x_{A} \mathbf{E}_{x}, \quad \mathbf{H}=I_{z z} \dot{\theta} \mathbf{E}_{z} \tag{7}
\end{equation*}
$$



Figure 1: A rigid body of mass $m$ is free to rotate about $A$ and is subject to a gravitational force and a torsional spring.
(a) (5 Points) Establish expressions for the angular momentum $\mathbf{H}_{A}$ and kinetic energy $T$ of the rigid body.
(b) (5 Points) Draw a free-body diagram of the rigid body.
(c) (8 Points) Using balances of linear and angular momentum, show that

$$
\begin{equation*}
\left(I_{z z}+m \ell^{2}\right) \ddot{\theta}+K \theta=m \ell\left(g \sin (\theta)+\ddot{x}_{A} \cos (\theta)\right) . \tag{8}
\end{equation*}
$$

(d) $(5$ Points) Starting from the work-energy theorem (5), prove that the total energy $E$ of the rigid body changes according to

$$
\begin{equation*}
\dot{E}=m \ddot{x}_{A} \dot{x}_{A}+? \tag{9}
\end{equation*}
$$

For full credit supply an expression for the total energy $E$ and the missing term.
(e) (2 Points) Suppose the truck is moving with constant speed $v_{0} \mathbf{E}_{x}\left(v_{0}>0\right)$ and the rigid body is initially at rest with $\theta=0^{\circ}$. In which direction will the rigid body tend to rotate if the truck suddenly brakes?

## Question 2 <br> Rolling of a Rigid Body (25 Points)

As shown in Figure 2, a semicircular cylinder of radius $R$, mass $m$, and moment of inertia (relative to its center of mass) $I_{z z}$ rolls on a cart. The position vectors of $A$ and the center of mass $C$ have the representations

$$
\begin{equation*}
\mathbf{x}_{A}=x \mathbf{E}_{x}+y_{0} \mathbf{E}_{y}, \quad \overline{\mathbf{x}}=\mathbf{x}_{A}-h \mathbf{e}_{y}, \tag{10}
\end{equation*}
$$

where $y_{0}$ and $h$ are constants. The point $P$ is the instantaneous point of contact of the semicylinder with the cart. The cart is in motion with respective velocity and acceleration vectors

$$
\begin{equation*}
\mathbf{v}_{\text {cart }}=v_{c} \mathbf{E}_{x}, \quad \mathbf{a}_{\text {cart }}=a_{c} \mathbf{E}_{x} \tag{11}
\end{equation*}
$$



Figure 2: A rigid semicylinder of mass $m$ and radius $R$ rolling on a moving cart.
(a) (5 Points) With the help of the identity $\mathbf{v}_{2}=\mathbf{v}_{1}+\boldsymbol{\omega} \times\left(\mathbf{x}_{2}-\mathbf{x}_{1}\right)$ applied to two points on the semicylinder, show that

$$
\begin{equation*}
\dot{x}+R \dot{\theta}=v_{c} \tag{12}
\end{equation*}
$$

where $\boldsymbol{\omega}=\dot{\theta} \mathbf{E}_{z}$ is the angular velocity of the semicylinder.
(b) ( 5 Points) Show that the total energy $E$ of the semicylinder can be expressed as a function of $v_{c}, \dot{\theta}$, and $\theta$.
(c) (3 Points) Draw a free-body diagram of the rolling rigid body.
(d) (8 Points) Using a balance of linear momentum, show that

$$
\begin{equation*}
\mathbf{F}_{f}+\mathbf{N}=m\left(a_{c} \mathbf{E}_{x}+? ?+? ? ?+? ? ? ?+g \mathbf{E}_{y}\right) . \tag{13}
\end{equation*}
$$

With the help of (13), show that the equation governing the motion of the semicylinder can be expressed as

$$
\begin{equation*}
\left(I_{z z}+m R^{2}+m h^{2}-2 m h R \cos (\theta)\right) \ddot{\theta}=-m R h \dot{\theta}^{2} \sin (\theta)-m g h \sin (\theta)+m a_{c}(? ? ? ? ?) . \tag{14}
\end{equation*}
$$

For full credit supply the missing terms in (13) and (14).
(e) (4 Points) Starting from the work-energy theorem (5), prove that the total energy $E$ of the rolling rigid body is not typically conserved.

## Question 3

Designing an Impacter
(25 Points)

As shown in Figure 3, an impacter in the form of a rigid body of mass $m$ and moment of inertia $I_{z z}$ is being designed. The impacter pivots from a pin-joint at point $O$. After being released from rest in the horizontal position, the point $S$ on the rigid body impacts the intended target when the body is vertical.


Figure 3: A rigid body of mass $m$ is pinpointed at a fixed point $O$. The body is released from rest and the point $S$ impacts with the rigid object show when the body's inclination is vertical.
(a) ( 7 Points) Starting from the following representation for the position of the center of mass $C$ relative to $O$,

$$
\begin{equation*}
\overline{\mathbf{x}}=d \mathbf{e}_{x}, \tag{15}
\end{equation*}
$$

establish expressions for the angular momentum $\mathbf{H}_{O}$, kinetic energy $T$, and total energy $E$ of the rigid body.
(b) (3 Points) Draw a free body diagram of the rigid body during the segment where it is in motion prior to the impact.
(c) (5 Points) Show that the equation of motion for the rigid body is

$$
\begin{equation*}
\left(I_{z z}+m d^{2}\right) \ddot{\theta}=-m g d \cos (\theta) . \tag{16}
\end{equation*}
$$

(d) (5 Points) If the body is released from rest at $\theta=0$, show that the velocity vector of the point $S$ on the body immediately prior to impact, which we denote by $\mathbf{v}_{S}^{*}$, is

$$
\begin{equation*}
\mathbf{v}_{S}^{*}=-\ell \sqrt{\frac{2 m g d}{I_{z z}+m d^{2}}} \mathbf{E}_{x} \tag{17}
\end{equation*}
$$

Here, $\ell$ is the distance that the point $S$ is from $O$ on the rigid body.
(e) (5 Points) How should $d$ be chosen so as to maximize the impact velocity $\left\|\mathbf{v}_{S}^{*}\right\|$ and $|\ddot{\theta}|$ ? Give a clear justification for your answer.

## Question 4 <br> A Swinging Bar (25 Points)

As shown in Figure 4, a uniform thin rod of mass $m$ and length $2 \ell$ is attached at one of its extremities to a fixed point $O$ by a linear spring of stiffness $K$ and unstretched length $L_{0}$. The bar is free to move on a smooth vertical plane.


Figure 4: A uniform rod of length $2 \ell$ is attached to a fixed point $O$ by a linear spring of stiffness $K$ and unstretched length $L_{0}$. The rod is free to move on a smooth vertical plane.

The center of mass $C$ of the rod has a position vector

$$
\begin{equation*}
\overline{\mathbf{x}}=x \mathbf{E}_{x}+y \mathbf{E}_{y} . \tag{18}
\end{equation*}
$$

The angular momentum of the rod relative to its center of mass $C$ is

$$
\begin{equation*}
\mathbf{H}=I_{z z} \dot{\theta} \mathbf{E}_{z} . \tag{19}
\end{equation*}
$$

(a) (5 Points) Show that the angular momentum $\mathbf{H}_{O}$ of the rod relative to $O$ has the representation

$$
\begin{equation*}
\mathbf{H}_{O}=I_{z z} \dot{\theta} \mathbf{E}_{z}+m(x \dot{y}-y \dot{x}) \mathbf{E}_{z} \tag{20}
\end{equation*}
$$

Establish an expression for the kinetic energy $T$ of the rod.
(b) (5 Points) Draw a free-body diagram of the rod. In your solution give a clear expression for the spring force.
(c) (5 Points) Show that equations of motion for the rod are

$$
\begin{equation*}
m \ddot{x}=? ?, \quad m \ddot{y}=-m g-? ? ? \quad I_{z z} \ddot{\theta}=? ? ? ? \tag{21}
\end{equation*}
$$

For full credit supply the missing terms.
(d) (5 Points) Starting from the work-energy theorem (5), prove that the total energy $E$ of the rod is constant. For full credit, give an expression for $E$.
(e) (5 Points) If gravity is ignored, which angular momentum of the rod is conserved? Clearly justify your answer.

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> A Moving Van (25 Points)

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## Solution to Question 2

Rolling of a Rigid Body (25 Points)

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## Question 3

Designing an Impacter ( 25 Points)

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