Solution of Midterm 1

ME 106, Fall 2015

Problem 1: (*Total 5 points*) This will not work. Suppose we have a perfect pump producing a perfect vacuum (absolute zero pressure). The pressure at the water level is atmospheric since the well is open. Using the hydrostatic relation between the pump (0) and water level (1)

$$P_0 = P_1 - \gamma h_{max}, \quad P_0 = 0, \quad \Rightarrow \quad h_{max} = \frac{P_0}{\gamma} = \frac{101000(\text{pa})}{998(\frac{\text{kg}}{\text{m}^3}) \times 9.81(\frac{\text{m}}{\text{s}^2})} = 10.31(\text{m}).$$

Thus a perfect pump will draw the water up only 10.31m, not the 18m required.

Problem 2: (Total 9 points) Bernoulli's equation represents conservation of energy or conservation of total pressure (1 point). The P is static or thermodynamics pressure (1 point), $\frac{1}{2}\rho V^2$ is dynamic pressure or specific kinetic energy (1 point), and ρgz is the hydrostatic pressure or specific potential energy due to gravity (1 point). The assumptions are

- Steady state or $\partial/\partial t = 0$. (1 point)
- Incompressible, or ρ is constant. (1 point)
- Inviscid, or $\mu = 0$. (1 point)
- No source of heat or isothermal (also no external work done to the flow). (1 point)
- Relation holds along a streamline or in 1D flow. (1 point)

Problem 3: (*Total 12 points*) Use Bernoulli's equation along a streamline from the top

surface (1) to the centerline of the exit jet(2).

$$P_{1} + \frac{1}{2}\rho V_{1}^{2} + \gamma z_{1} = P_{2} + \frac{1}{2}\rho V_{2}^{2} + \gamma z_{2}, \quad (3 \text{ point})$$
$$P_{1} = P_{atm}, z_{1} - z_{2} = H, V_{1} = 0 \quad (3 \text{ point})$$

Surface tension around the jet increases the pressure inside the jet (P_2) according to

$$P_2 - P_{atm} = \frac{2\sigma}{D}$$
 (3 point)

substituting the above into Bernoulli's equation and solving for V_2 we obtain

$$V_2 = \sqrt{\frac{1}{\rho} \left(\frac{-4\sigma}{D} + 2\gamma H\right)} \quad (3 \text{ point})$$

Problem 4: (*Total 20 points*) Let b = 5(ft) be the width of the gate into the page. The



resultant pressure on the left side is the pressure at the centroid $h_{1c} = 5(\text{ft})$

$$P_{1c} = \gamma_w h_{1c} = \gamma(\text{lb/ft}^3) \times 5(\text{ft}) = 5\gamma(\text{lb/ft}^2), \quad (2 \text{ point})$$

The resultant force F_1 is pressure P_{1c} times wet area A_1

$$A_{1} = h_{1}b = 10(\text{ft}) \times 5(\text{ft}) = 10b(\text{ft}^{2}), \quad (1 \text{ point})$$
$$F_{1} = P_{1c}A_{1} = 5\gamma(\text{lb}/\text{ft}^{2}) \times 10b(\text{ft}^{2}) = 50\gamma b(\text{lb}). \quad (2 \text{ point})$$

The resultant force F_1 acts at the centroid of the pressure prism (2/3 depth = 20/3 ft). This can also be derived as

$$y_{1p} = y_{1c} + \frac{I_{xx}^c}{y_{1c}A_1} = \frac{h_1}{2} + \frac{\frac{1}{12}h_1^3b}{\frac{h_1}{2}(h_1b)} = \frac{2}{3}h_1 = \frac{2}{3} \times 10 = 6.67(\text{ft}).$$
 (3 point)

Hence the moment arm about the hinge is $L_1 = 2(\text{ft}) + 6.67(\text{ft}) = 8.67(\text{ft})$ (1 point).

On the right side, the centroid is at $h_{2c} = h/2$, and the pressure at centroid is P_{2c} , the wet area A_2 and the force from the right side F_2 are

$$P_{2c} = (SG)\gamma(h/2) = 1.025 \times \gamma(\text{lb/ft}^3)h/2 = 0.5125\gamma h(\text{lb/ft}^2), \quad (2 \text{ point})$$

$$A_2 = hb(\text{ft}^2), \quad (1 \text{ point})$$

$$F_2 = P_{2c}A_2 = 0.5125\gamma h(\text{lb/ft}^2)hb(\text{ft}^2) = 0.5125\gamma bh^2(\text{lb}). \quad (2 \text{ point})$$

The resultant force F_2 acts at 2/3 depth (2h/3 ft), which can also be derived as

$$y_{2p} = y_{2c} + \frac{I_{xx}^c}{y_{2c}A_2} = \frac{h}{2} + \frac{\frac{1}{12}h^3b}{\frac{h}{2}(hb)} = \frac{2}{3}h(\text{ft}).$$
 (2 point)

Hence the moment arm from point A is $L_2 = 12(\text{ft}) - h/3(\text{ft})$ (1 point).

The balance of moments around point A is

$$\sum M_A = 0, \quad \Rightarrow \quad F_1 L_1 - F_2 L_2 = 0,$$

50\gap b(lb) \times 8.67(ft) = 0.5125\gap bh^2(lb) \times (12 - h/3)(ft) (2 point)

Simplifying and solving $h^2(36 - h) = 2537.56$ yields h = 9.85(ft) (1 point).

Problem 5: (Total 20 points) (a) From conservation of mass $Q = V_A A_A = V_B A_B$ (2 point). The area of the pipe outlet is $A_A = \frac{\pi}{4}D^2 = 7.8538 \times 10^{-5} (\text{m}^2)$ (1 point) and the area of horizontal output at the edge of circular disk is $A_B = 2\pi Rt = 1.2566 \times 10^{-4} (\text{m}^2)$ (1 point). Writing Bernoulli between point A and B (1 point)

$$\frac{P_A}{\gamma} + \frac{V_A^2}{2g} + z_A = \frac{P_B}{\gamma} + \frac{V_B^2}{2g} + z_B.$$



Since both points A and B are free jets, $P_A = P_B = P_{atm}$ (1 point) and the cancel from the equation. Also set $z_A = 0$ and $z_B = z$ (1 point), the elevation of point B with respect to the pipe outlet. Moreover, from conservation of mass

$$V_A = \frac{Q}{A_A}, \quad V_B = \frac{Q}{A_B}.$$
 (2 point)

Substituting velocities in Bernoulli yields

$$\frac{Q^2}{2gA_A^2} = \frac{Q^2}{2gA_B^2} + z.$$
 (1 point)

Solving for Q gives (2 point)

$$Q = \sqrt{\frac{2gz}{A_A^{-2} - A_B^{-2}}} = \sqrt{\frac{2 \times 9.81 (\text{m/s}^2) \times 0.2 (\text{m})}{(7.8538 \times 10^{-5} (\text{m}^2))^{-2} - (1.2566 \times 10^{-4} (\text{m}^2))^{-2}}} = 2 \times 10^{-4} (\text{m}^3/\text{s}).$$

(b) Writing Bernoulli equation between point (A) and (C)

$$\frac{P_A}{\gamma} + \frac{V_A^2}{2g} + z_A = \frac{P_C}{\gamma} + \frac{V_C^2}{2g} + z_C.$$
 (1 point)

The flow at (A) is free jet so the gage pressure $P_A = P_{atm} = 0$ (1 point). However, point (C) is stagnation point, hence P_C is the stagnation pressure, and $V_C = 0$ (1 point). The height of manometer is $H = P_C/\gamma$ (2 point), so

$$H = \frac{P_C}{\gamma} = \frac{V_A^2}{2g} - z. \quad (1 \text{ point})$$

Also $V_A = Q/A_A = 2 \times 10^{-4} (\text{m}^3/\text{s})/7.8538 \times 10^{-5} (\text{m}^2) = 2.545 (\text{m/s})$ (1 point). Thus

$$H = \frac{(2.545(\text{m/s}))^2}{2 \times 9.81(\text{m/s}^2)} - 0.2(\text{m}) = 0.13(\text{m}).$$
 (1 point)