Problem 1. (33 points total)

$$v_T = \frac{10.07 \text{ m/s}}{3.6} = 10.07 \text{ m/s}$$

 $v_C = \frac{80}{3.6} = 22.22 \text{ m/s}$

 $\frac{60}{2}$ = 16.67 m/s (14 points for relative velocity)

Attach a translating xy-frame to the car C with the x-axis along the normal direction of the truck T. The same unit vectors **i**, **j** can be used in both the translating and fixed frames. Thus

$$\mathbf{v}_{T} = \mathbf{v}_{C} + \mathbf{v}_{T/C}$$

$$\Rightarrow \quad 16.67\mathbf{j} = 22.22\cos 30^{\circ}\mathbf{i} + 22.22\sin 30^{\circ}\mathbf{j} + \mathbf{v}_{T/C}$$

$$\Rightarrow \quad \mathbf{v}_{T/C} = -19.24\mathbf{i} + 5.56\mathbf{j} \text{ m/s} \qquad (3 \text{ points for answer})$$

The solution may also be obtained graphically from the vector diagram of velocities. In comparison, the absolute velocity of truck T is

$$v_T = 16.67 j$$



(14 points for relative acceleration)

Similarly,

$$a_T = \frac{v_T^2}{\rho} = \frac{16.67^2}{110} = 2.53 \,\mathrm{m/s^2}$$

 $a_T = 1.5 \,\mathrm{m/s^2}$

The acceleration of *T* relative to *C* is given by

$$\mathbf{a}_{T} = \mathbf{a}_{C} + \mathbf{a}_{T/C}$$

$$\Rightarrow 2.53\mathbf{i} = 1.5\cos 30^{\circ}\mathbf{i} + 1.5\sin 30^{\circ}\mathbf{j} + \mathbf{a}_{T/C}$$

$$\Rightarrow \mathbf{a}_{T/C} = 1.23\mathbf{i} - 0.75\mathbf{j} \text{ m/s}^{2} \qquad (3 \text{ points for answer})$$

$$a_{C} = 1.5 \qquad 30^{\circ} \qquad 31.2^{\circ} \qquad a_{T/C} = 1.45$$

(b) (5 points) A coordinate system attached to the truck *T* (with the *y*-axis in the direction of the velocity of *T* for example) is a rotating system. If \mathbf{v}_{rel} is the velocity of car *C* as observed from truck *T*, then \mathbf{v}_{rel} depends on the rotation of coordinate system attached to the truck. It can be shown from rigid-body kinematics that

$$\mathbf{v}_{rel} = \mathbf{v}_C - \mathbf{v}_T - \mathbf{\omega} \times \mathbf{r}_{C/T}$$
(this is not expected)

$$\Rightarrow \mathbf{v}_{rel} \neq \mathbf{v}_C - \mathbf{v}_T = -\mathbf{v}_{T/C}$$
(3 points for answer)

Problem 2. (33 points total)

Attach *xy*-frame to the center of the upper pulley. Locate *A*, *B* by coordinates x_A and y_B . Since $x_A \le 0$,

$$-x_A + 2y_B = L \qquad \Rightarrow \qquad v_A = 2v_B \qquad (7 \text{ points for constraint})$$

$$\Rightarrow \qquad v_A(0) = 2v_B(0) = 6 \text{ ft/sec}$$

For block A,

(10 points for block A)

(10 points for block *B*)

For block *B*,

$$\int \sum F_{y} dt = \Delta G_{y} \qquad \Rightarrow \qquad \int_{0}^{1} (-2T + m_{B}g) dt = m_{B}v_{B}(1) - m_{B}v_{B}(0) \qquad \downarrow$$
$$\Rightarrow \qquad -2T + 3 = \frac{3}{32.2} \left[\frac{v_{A}(1)}{2} - 3 \right] \qquad (2)$$

There are 2 unknowns $v_A(1)$, T in 2 equations. Simultaneous solution gives



Problem 3. (34 points total)

The solution is divided into three parts so that the impact between A and B can be treated separately.

Part 1. (10 points) Let v_A be the velocity with which the block A hits the pan B.

$$\Delta T + \Delta V_g = 0 \implies \qquad m_A g h = \frac{1}{2} m_A v_A^2$$
$$\implies \qquad v_A = 6.26 \text{ m/s}$$

Part 2. (10 points) Let v be the common velocity of the block A and pan B after the plastic impact. Then

$$\Delta G = 0 \qquad \Rightarrow \qquad m_A v_A + m_B v_B = (m_A + m_B) v$$

$$\Rightarrow \qquad 30(6.26) = (30 + 10) v$$

$$\Rightarrow \qquad v = 4.70 \text{ m/s} \qquad \downarrow$$

Part 3. (14 points) Let δ be the maximum deflection of the pan measured from its initial level of static equilibrium. The initial compression of the spring is

 $e = m_B g / k = 4.91 \times 10^{-3} \text{ m}$ For the system consisting of *A* and *B*, $U_{1-2} = \Delta T = -T_1$ $(m_A + m_B)g\delta - \int_e^{e+\delta} kx dx = -\frac{1}{2}(m_A + m_B)v^2$ \Rightarrow $\Rightarrow \qquad 40(9.81)\delta - 10000[(e+\delta)^2 - e^2] + 20v^2 = 0$ $10000\delta^2 - 294.2\delta - 441.8 = 0$ \Rightarrow $\delta = 0.225$ or -0.196 \Rightarrow Take the positive root to write (5 points for answer) $\delta = 225 \text{ mm}$ kx Position 1: at reference level Position 2: δ below $(m_A + m_B)g$ reference