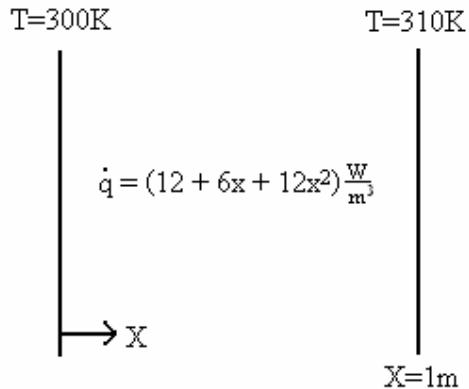


Problem 1 (40)



Goal: Find q_{total}

Solution:

Fastest way

$$\text{Total heat generated} = \text{Total heat lost} = q$$

$$q = \int \dot{q} dV = A_c \int_0^1 \dot{q} dx$$

$$q = A_c \int_0^1 (12 + 6x + 12x^2) dx$$

$$q = 1 \left[12x + 3x^2 + 4x^3 \right]_0^1$$

$$q = 19W$$

Standard analytical way

$$k \frac{d^2T}{dx^2} + \dot{q} = 0$$

$$\frac{d^2T}{dx^2} = -\frac{\dot{q}}{k} = -\frac{12 + 6x + 12x^2}{k}$$

$$\frac{dT}{dx} = -\frac{12x + 3x^2 + 4x^3}{k} + C_1$$

$$T(x) = -\frac{6x^2 + x^3 + x^4}{k} + C_1 x + C_2$$

Apply BC

$$300 = T(0) = C_2$$

$$310 = T(1) = -\frac{6+1+1}{1} + C_1 + 300$$

$$C_1 = 18$$

$$T(x) = -\frac{6x^2 + x^3 + x^4}{k} + 18x + 300$$

$$T(x) = -x^4 - x^3 - 6x^2 + 18x + 300$$

Calculate flux

$$q(x) = -kA \frac{dT}{dx} = -(-4x^3 - 3x^2 - 12x + 18) = 4x^3 + 3x^2 + 12x - 18$$

$$q_{total} = -q(0) + q(1) = 18 + (4 + 3 + 12 - 18)$$

$$q_{total} = 19W$$

Finite Difference method

1 node

T surf not usable

Entire wall is control volume, so $q_{net_in} = -q_{generated}$

$q_{generated}$ – if evaluated as integral, exact answer

if evaluated at midpoint $(12 + 6*(1/2) + 12*(1/4)) = 18W$

2 node

If split the difference, T surf not usable

If one on each boundary, , not useful

3 node

Can solve for T_{middle}

$$q_{in} - q_{out} + q_{gen} = kA \frac{(300 - T_{middle})}{0.5} - kA \frac{(T_{middle} - 310)}{0.5} + \frac{18}{2} = 0$$

$$2T_{middle} = 300 + 310 + \frac{18}{4}$$

$$T_{middle} = \frac{614.5}{2} = 307.25$$

Then look at each side

Left, note volume = 1/4

$$q_{in} - q_{out} + q_{gen} = 0$$

$$q_{in} = q_{out} - q_{gen} = kA \frac{300 - T_{middle}}{0.5} - \frac{12}{4}$$

$$q_{in} = \frac{300 - 307.25}{0.5} - 3 = -17.5$$

Right, note volume = 1/4

$$q_{in} - q_{out} + q_{gen} = 0$$

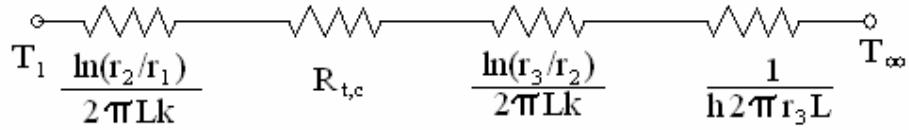
$$q_{out} = q_{in} + q_{gen} = kA \frac{307.25 - 310}{0.5} + \frac{30}{4} = 2$$

Sum

$$q_{out_total} = q_{right_out} - q_{left_in} = 19.5W$$

Problem 2 (40)

Draw a circuit between inner radius and air

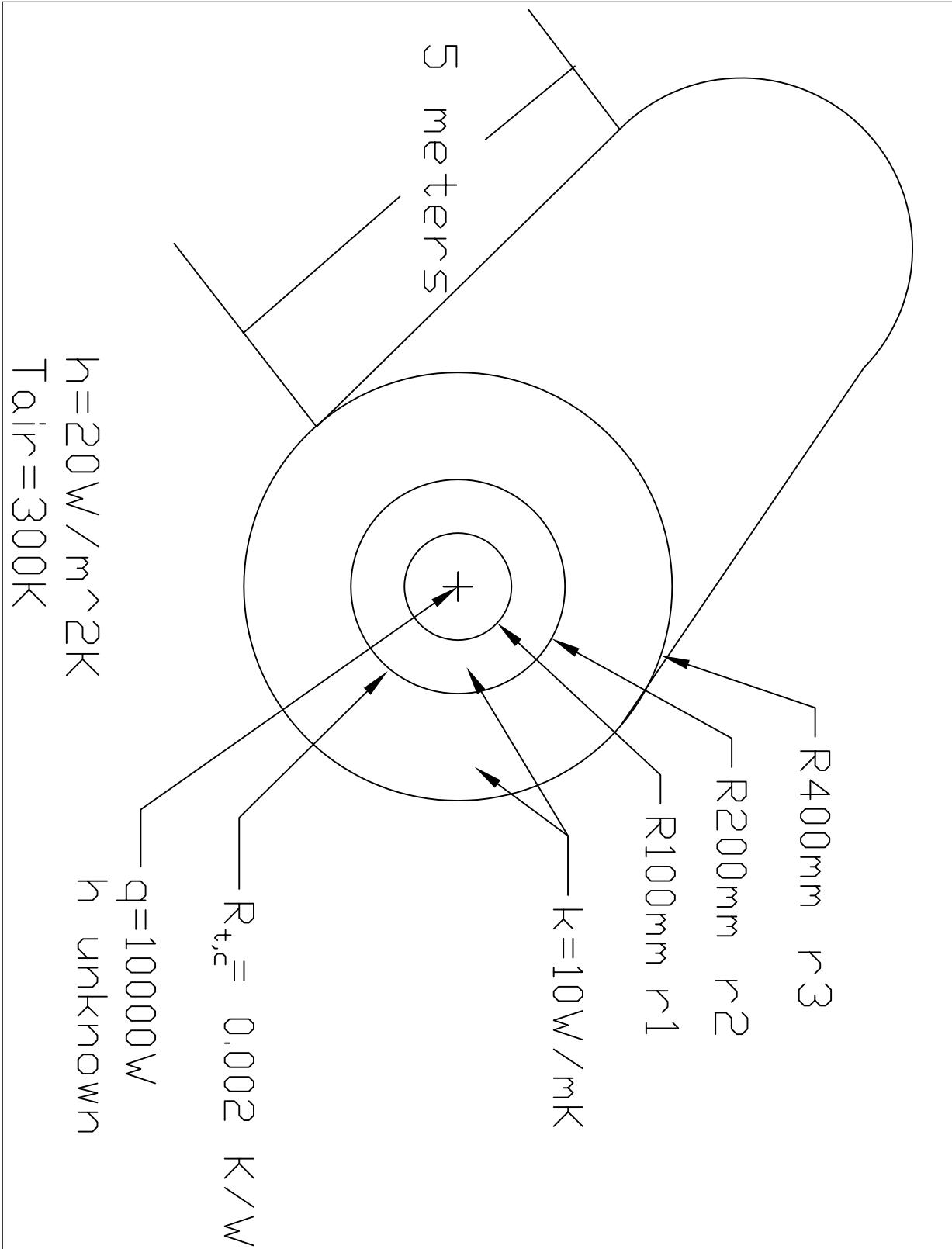


$$q = \frac{T_1 - T_\infty}{\frac{\ln(r_2/r_1)}{2\pi L k} + R_{t,c} + \frac{\ln(r_3/r_2)}{2\pi L k} + \frac{1}{h 2\pi r_3 L}}$$

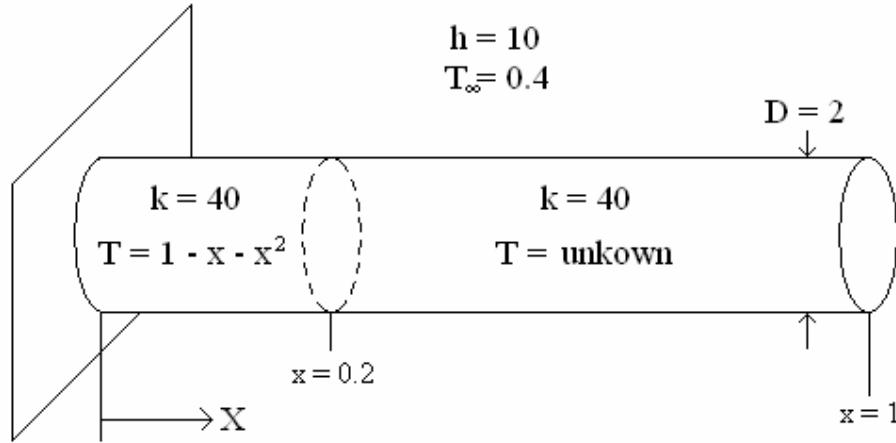
$$T_1 = q \left[\frac{\ln(r_2/r_1)}{2\pi L k} + R_{t,c} + \frac{\ln(r_3/r_2)}{2\pi L k} + \frac{1}{h 2\pi r_3 L} \right] + T_\infty$$

$$T_1 = 10000W \left[\frac{\ln\left(\frac{0.2m}{0.1m}\right)}{2\pi(5m)\left(10\frac{W}{mK}\right)} + 0.002K/W + \frac{\ln\left(\frac{0.4m}{0.2m}\right)}{2\pi(5m)\left(10\frac{W}{mK}\right)} + \frac{1}{\left(20\frac{W}{m^2K}\right)2\pi(0.4m)(5m)} \right] + 300K$$

$$T_1 = 403.916 \text{ K}$$



Problem 3 (30)



Goal: Find the steady state heat loss from fin region $x=0.2$ to $x=1$

Solution

Drawing a control volume around $x=0.2$ to $x=1$, $q_{in} = q_{out}$.

$$q_{in} = -kA \frac{dT}{dx} \Big|_{x=0.2^+}$$

q should be continuous across the boundary

$$q_{in} = -kA \frac{dT}{dx} \Big|_{x=0.2^+} = -kA \frac{dT}{dx} \Big|_{x=0.2^-}$$

Plug in T

$$q_{in} = -k \left(\frac{\pi D^2}{4} \right) \frac{d(1-x-x^2)}{dx} \Big|_{x=0.2^-}$$

$$q_{in} = -\frac{k\pi D^2}{4} (-1-2x) \Big|_{x=0.2}$$

$$q_{in} = -\frac{k\pi D^2}{4} (-1-0.4)$$

$$q_{in} = \frac{1.4k\pi D^2}{4}$$

$$q_{in} = \frac{1.4(40)\pi(2)^2}{4} = 175.9$$

$$q_{out} = 175.9$$